

Machine Design Course for Communication / Electrical Department

Sheet 2 Solution - Design of Bolts

Problem 1

Two plates 10 mm in thicknesses and subjected to a tensile load of $F = 4000$ N are connected by 4 bolts as shown in Figure 1. Compute the diameter of the bolts if the maximum stress in the bolts is 200 MPa

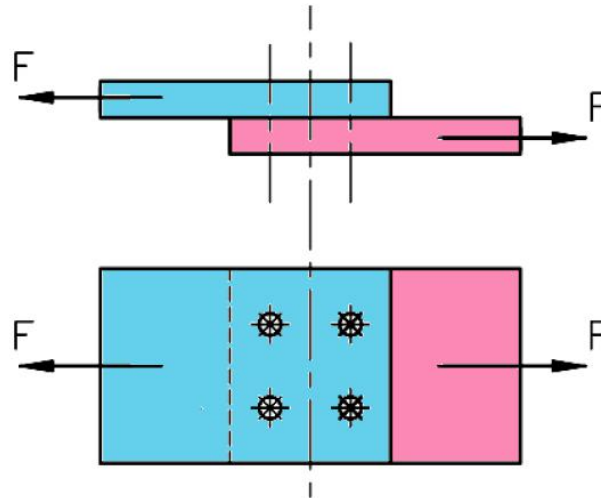


Figure 1.

Givens:

$t=10$ mm $F=4000$ N $N=4$ bolts $\tau_{bolts\ max}=200$ Mpa

Req: $d=?$

Solution

$$\tau_{bolts} = \frac{F}{n \times Area} \leq \tau_{bolts, max}$$

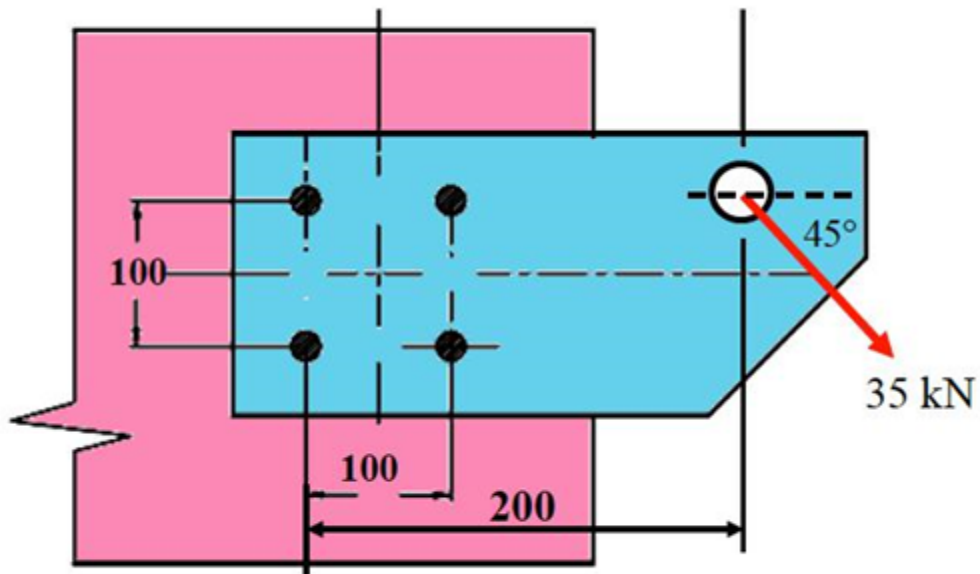
$$\frac{4000}{4 \times \frac{\pi}{4} d_i^2} \leq 200$$

$$d_i = 6.4 \text{ mm} \quad \rightarrow \quad d_o = d_i / 0.85 = 7.5 \text{ mm}$$

from table the type of bolts will be M8

Problem 2

Four bolts are used to secure the bracket to the wall as shown in Figure 2. All the dimensions are in millimeter. If the bolts are made of the steel having $S_y=420$ MPa, determine their size of bolts using factor of safety of 2.



Given:

$S_y=420$ Mpa f.o.s = 2 $N=4$

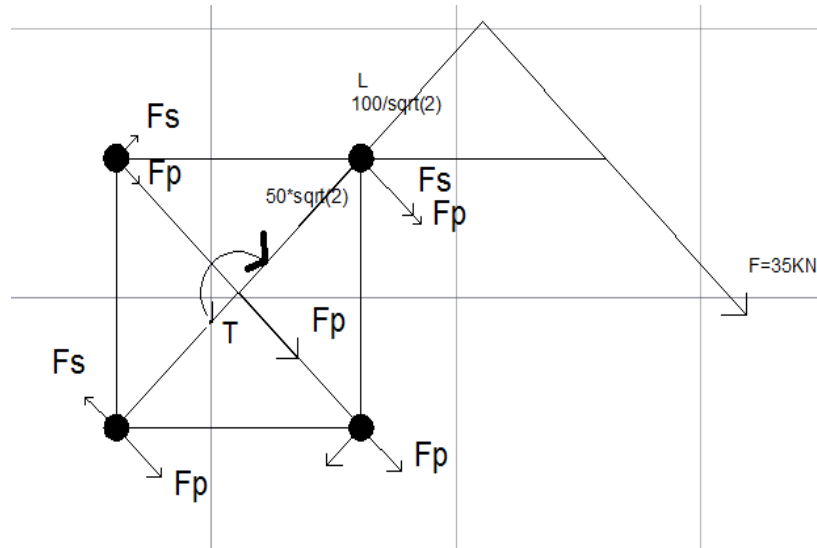
Req:

Size of bolts

Solution

Primary stress:

$$\tau_p = \frac{F}{n \times Area} = \frac{35 \times 1000}{4 \times Area} = 8.75 \times 10^3 / A \rightarrow 1$$



Secondary stress:

$$T = F \cdot L = F_1 \cdot r_1 + F_2 \cdot r_2 + F_3 \cdot r_3 + F_4 \cdot r_4$$

$$F_1/r_1 = F_2/r_2 = F_3/r_3 = F_4/r_4$$

$$\text{As: } r_1 = r_2 = r_3 = r_4$$

$$\text{So: } F_1 = F_2 = F_3 = F_4$$

$$T = F \cdot l = 4 \cdot F_s \cdot r \quad \rightarrow \quad F_s = 17.5 \cdot 10^3 \text{ N}$$

$$\tau_s = \frac{F_s}{\text{Area}} = \frac{17.5 \times 1000}{\text{Area}} \text{ at min } \theta \text{ or } r_{\max}$$

So the max stress at bolt no. 1 $\theta = 0.0^\circ$

$$\tau_{tot} = \sqrt{\tau_p^2 + \tau_s^2 + 2 \times \tau_p \times \tau_s \times \cos \theta}$$

$$\tau_{tot} = \frac{(8.75 \times 1000 + 17.5 \times 1000)}{\text{Area}} = \frac{26.25 \times 1000}{\text{Area}}$$

$$\tau_{tot} \leq \frac{S_y}{f.o.s}$$

$$\frac{26.25 \times 1000}{\text{Area}} \leq 0.577 \times \frac{420}{2}$$

$$\text{Area} = 250 \text{ mm}^2$$

$$d_i = 17.8 \text{ mm}$$

$$d_o = d_i / 0.85 = 20.9 \text{ mm}$$

from table we take M24

Problem 3

Three bolts are used to secure the bracket to the wall as shown in Figure 3. All the dimensions are in millimeter. If the bolts are made of the steel having $S_y=380$ MPa, determine their size of bolts using factor of safety of 2.5.

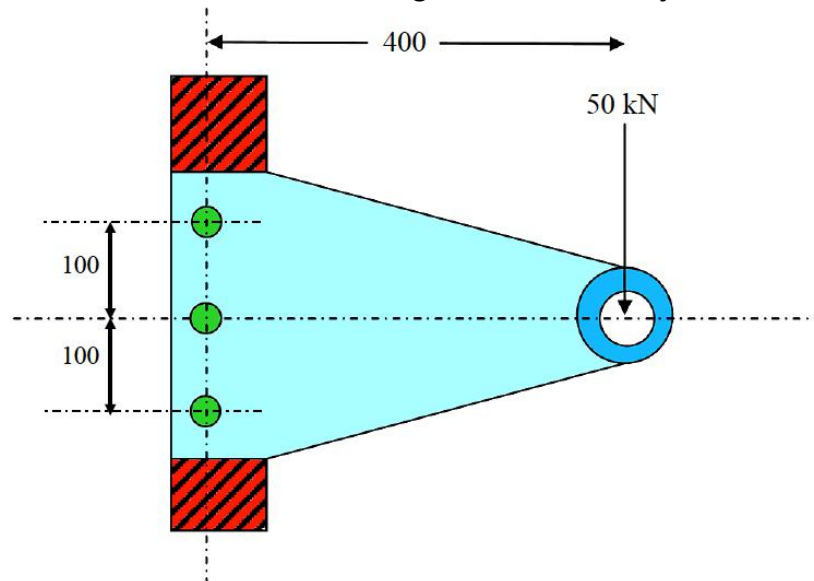


Figure 3.

Givens:

$S_y=380$ MPa $n=2.5$ $N=3$ $F=50$ kN

Req:

Size of bolts?

Solution

Primary stress:

$$\tau_p = \frac{F}{n \times Area} = \frac{5000}{3 \times Area} = 16.667 \times 10^3 / A \rightarrow 1$$

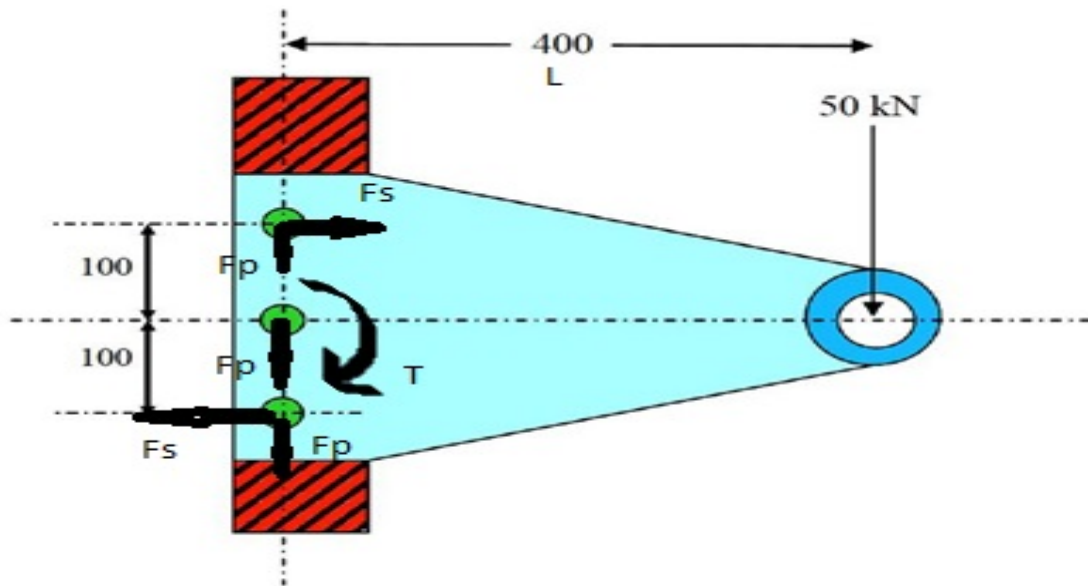


Figure 3.

Secondary stress:

$$T = F \cdot L = F_1 \cdot r_1 + F_2 \cdot 0 + F_3 \cdot r_3$$

$$F_1 / r_1 = F_3 / r_3$$

$$\text{As: } r_1 = r_3$$

$$\text{So: } F_1 = F_3$$

$$T = F \cdot 400 = 2 \cdot F_s \cdot r \quad \rightarrow \quad F_s = 100 \cdot 10^3 \text{ N}$$

$$\tau_s = \frac{F_s}{\text{Area}} = \frac{100 \times 1000}{\text{Area}} = \rightarrow 2$$

Max total stress at min Θ or r_{\max}

So the max stress at bolt no. 1 or 3 $\Theta = 90^\circ$

$$\tau_{tot} = \sqrt{\tau_p^2 + \tau_s^2 + 2 \times \tau_p \times \tau_s \times \cos \theta}$$

$$\tau_{tot} = \frac{\sqrt{(16.667 \times 10^3)^2 + (100 \times 10^3)^2 + 2 \times 16.667 \times 10^3 \times 100 \times 10^3 \times \cos 90}}{\text{Area}}$$

$$\tau_{tot} = \frac{101.38 \times 1000}{\text{Area}}$$

$$\tau_{tot} \leq \frac{S_y}{f.o.s}$$

$$\frac{101.38 \times 1000}{Area} \leq 0.577 \times \frac{380}{2.5}$$

$d_i = 38.36 \text{ mm}$

$d_o = d_i / 0.85 = 45.13 \text{ mm}$

→ from table we take M48

Problem 4

The bracket shown in Figure 4 is secured to a 'C' column by means of three M16 through bolts having $S_y=620$ MPa, the bracket is subjected to vertical load of 16 kN. Determine the factor of safety for the bolts. Neglect the stresses due to initial tension in bolts. All the dimensions are in millimeter.

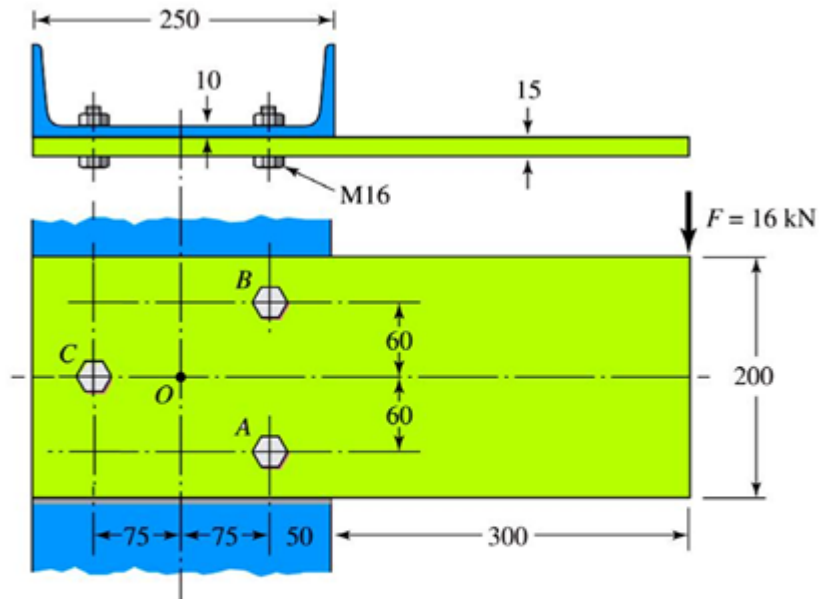


Figure 4.

2

Givens:

$S_y=620$ MPa $N=3$ $F=16$ kN bolt size=M16

Req:

$n=??$

solution

Primary stress:

$$\tau_p = \frac{F}{n \times Area} = \frac{16000}{3 \times \frac{\pi}{4} d^2} = \frac{16000}{3 \times \frac{\pi}{4} (0.85 \times 16)^2} = 145.3 \text{ Mpa} \rightarrow 1$$

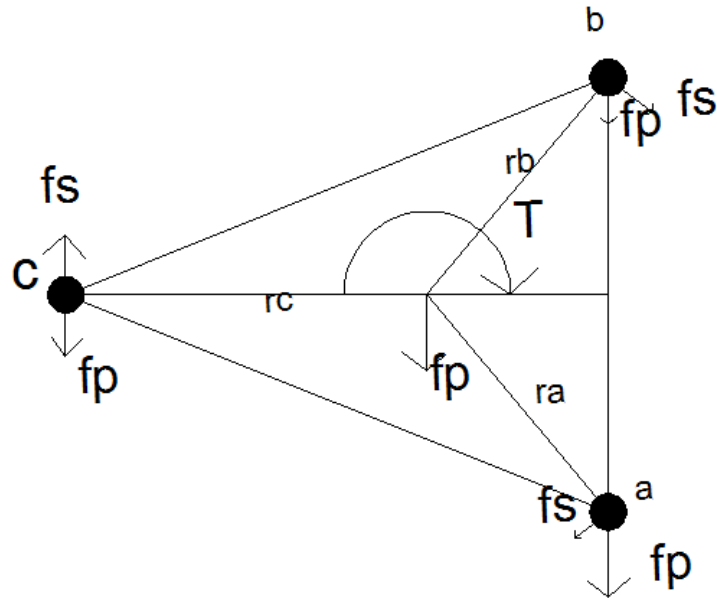
Secondary stress:

$$T = F \cdot 400 = F_a \cdot r_a + F_b \cdot r_b + F_c \cdot r_c$$

$$F_a / r_a = F_b / r_b = F_c / r_c$$

As: $r_c=100\text{mm}$ $r_a=r_b=\sqrt{50^2+60^2}=78.1\text{mm}$
 So: $F_a=F_b=22515.3\text{ N}$ $F_c=28829\text{ N}$

$$\tau_{s,a\text{ or }b} = 22515.3 / (3 * (\pi * (0.85 * 16)^2) / 4) = 155\text{ Mpa}$$



$$\tau_{s,c} = 28829 / (3 * (\pi * (0.85 * 16)^2) / 4) = 198.5\text{ Mpa}$$

Max total stress at Θ_{\min} or r_{\max}
 max r at c min Θ at a or b

Bolt a:

$$\cos \theta = 50 / 78.1 = 0.64$$

$$\tau_{a,tot} = \sqrt{\tau_p^2 + \tau_s^2 + 2 * \tau_p * \tau_s * \cos \theta}$$

$$\tau_{a,tot} = \frac{\sqrt{(16.667 * 10^3)^2 + (100 * 10^3)^2 + 2 * 16.667 * 10^3 * 100 * 10^3 * \cos 90}}{\text{Area}}$$

$$\tau_{tot} = 181\text{ Mpa}$$

Bolt c:

$$\Theta_c = 180^\circ$$

$$\tau_{c,tot} = \sqrt{\tau_p^2 + \tau_s^2 + 2 \times \tau_p \times \tau_s \times \cos \theta}$$

$$\tau_{c,tot} = 162 \text{ Mpa}$$

$$\tau_{max,tot} = 181 \text{ Mpa}$$

Design Eqn:

$$\tau_{max,tot} \leq \frac{S_y}{f.o.s}$$
$$181 \leq 0.577 \times \frac{620}{f.o.s}$$

$$\rightarrow f.o.s = 1.96$$